



POWER GENERATION THROUGH VIBRATION- BASED ENERGY HARVESTERS

- a) Hybrid piezo-electromagnetic energy harvester
- b) Electromagnetic energy harvesters' array

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Outline

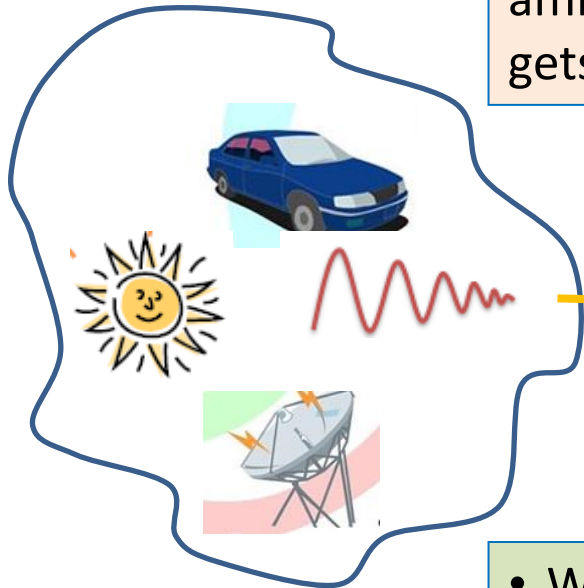
- ❖ Introduction
- ❖ Literature Surveys
- ❖ Motivation and Objectives
- ❖ Investigation of hybrid harvester
- ❖ Demonstration of hybrid power application
- ❖ Investigation of an array of electromagnetic energy harvester
- ❖ Conclusion

Introduction and Literature Survey

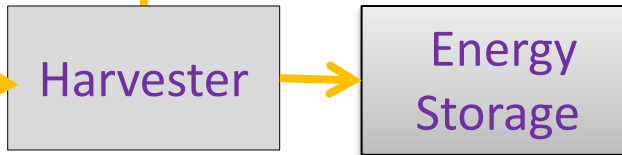
Energy Harvesting

What is energy harvesting?

Extracting electrical energy from ambient resources which otherwise gets wasted.



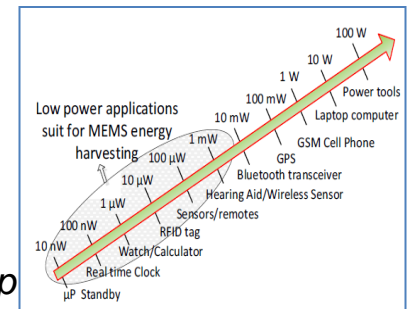
Ambient Energy



- Wireless applications.
- Condition monitoring: Bridge, Building, Traffic data, Health etc.,
- Major role: Smart homes, Smart buildings and Smart cities



Low power devices

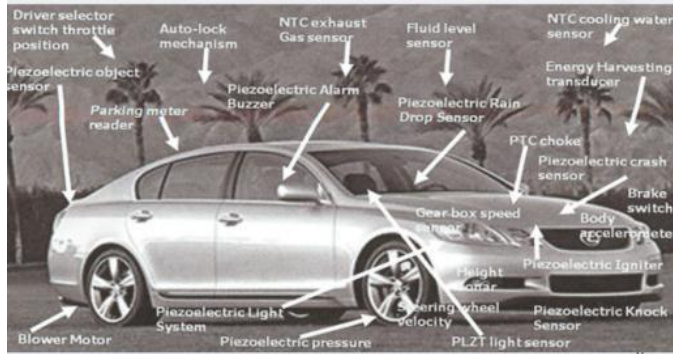


micro-energy cell (MEC) product overview *Tech. Rep*

Applications of energy harvesting

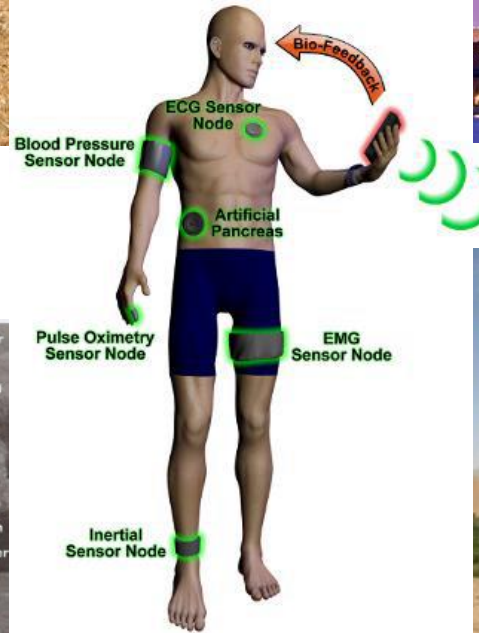


Environmental Monitoring



Automobile applications

Wireless Sensor Networks (WSNs)



Medical remote sensing



Structural Monitoring

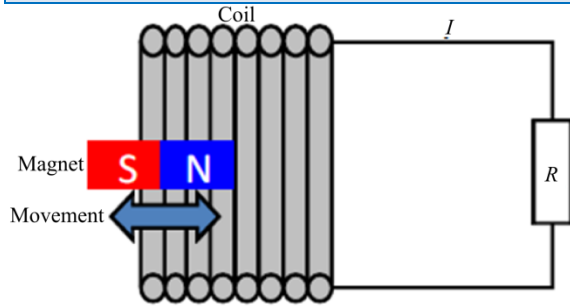


Military applications

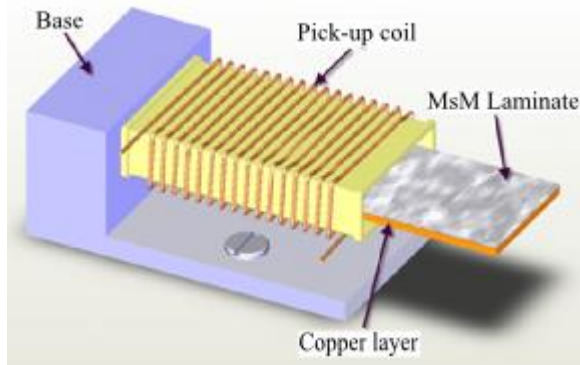
Almost 90% of WSNs applications cannot be enabled without Energy Harvesting technologies that allow self-powering features- F Cottone

Vibration Energy Harvesters - Transduction Techniques

Electromagnetic: Any change in the magnetic field of a solenoid will induce an emf in the solenoid.



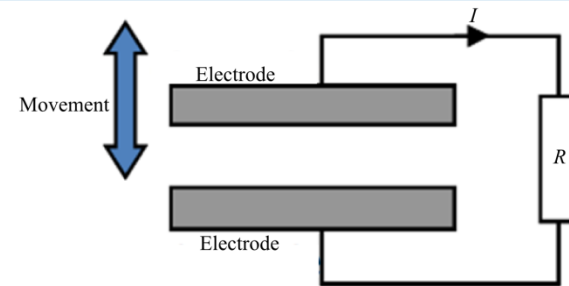
Magnetostrictive: Generates power because of change in magnetization due to applied stress.



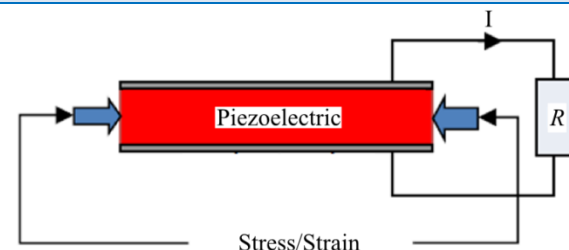
Boisseau et al., 2012

Why Vibration?: Commonly available in structural and industrial environments.

Electrostatic: Generates power when the transducer moves against an electrical field.



Piezoelectric: Generates power when mechanical stress is applied to the piezoelectric elements through external vibrations.



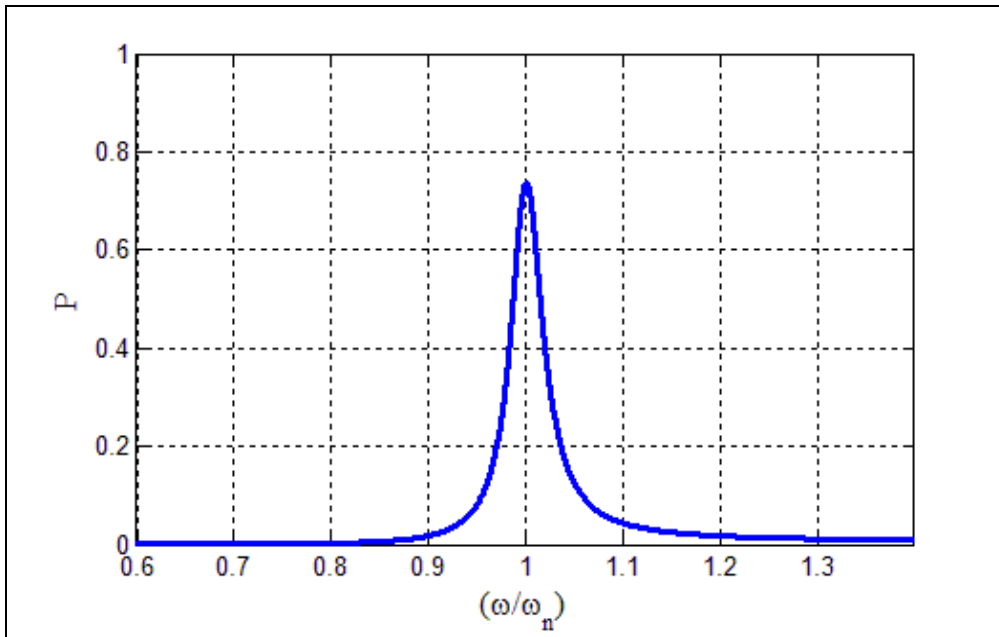
Literature Survey

- Piezoelectric EH: High frequency
High voltage
Low current
- Electromagnetic EH: Low frequency
Low voltage
High current

Need of Hybrid Energy harvesting (Tadesse *et al.*, 2009, Khaligh *et al.*, 2010)

Why Broadband Energy harvesting?

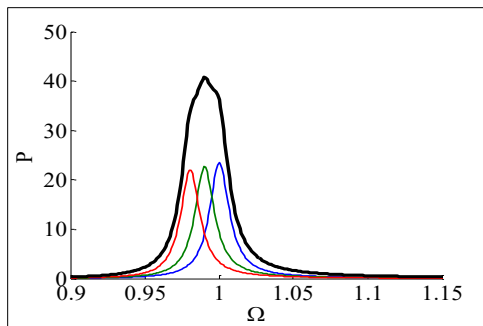
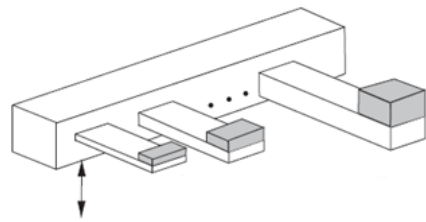
- Standalone linear harvester: Harvested power reduces drastically as the excitation frequency moves away from the resonant frequency.
- Ambient vibrations are random in nature; thus, these harvesters perform poorly under such excitations.



Broadband energy harvesting

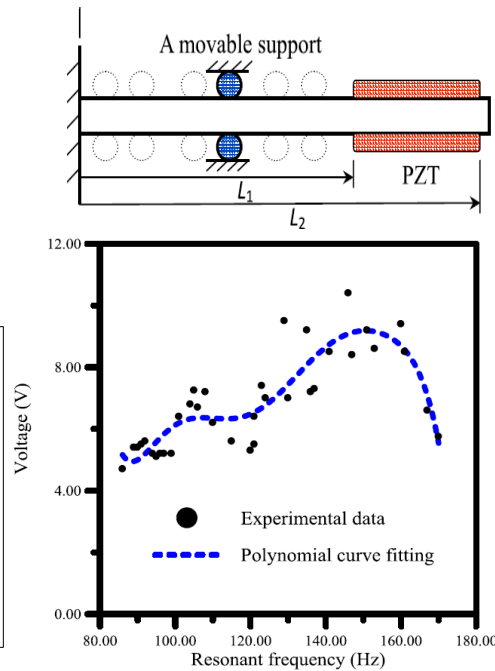
Increasing the operating bandwidth so that fairly large power output is obtained over a range of frequencies.

Multi-frequency technique



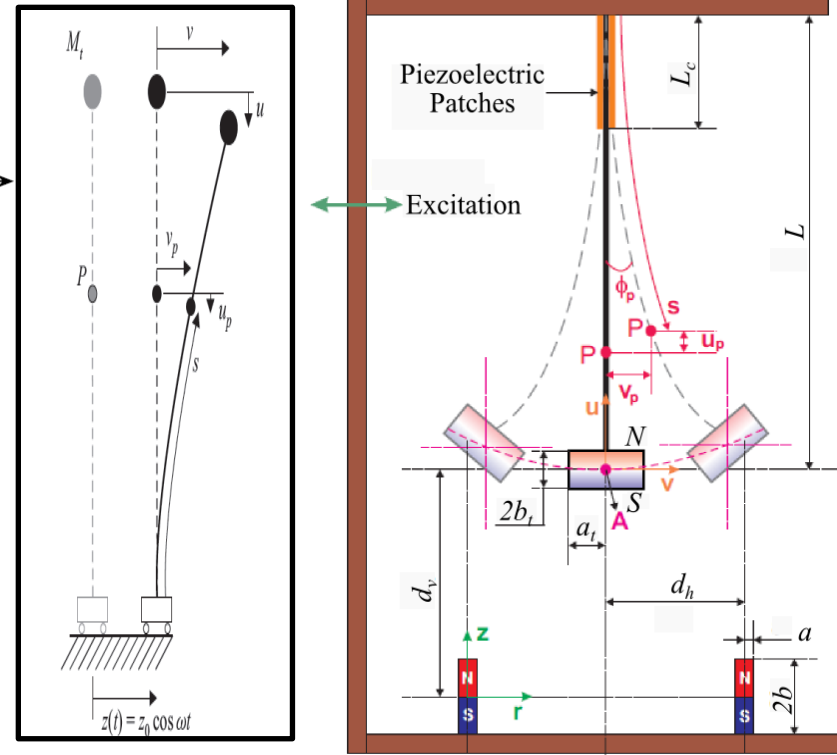
S.M. Shahruz (2008)

Resonance tuning technique



Huang and Lin, 2012

Nonlinear harvesting technique



Friswell et al., 2012, Kumar et al., 2015

Motivation and Objectives

Motivation :

- To scavenge a reasonable amount of power over a broad frequency range using:
 - (i) Hybrid transduction mechanism
 - (ii) Electromagnetic energy harvesters' array.

Objectives:

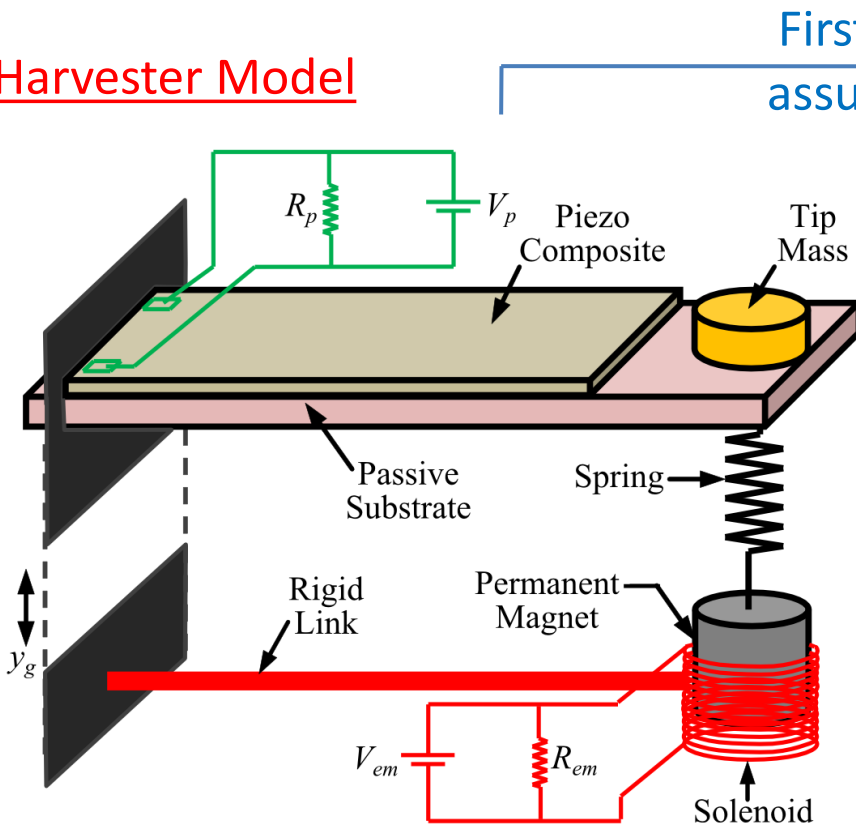
- Analysis of the hybrid energy harvester under harmonic and random excitations and comparing with the standalones.
- Verification of the results through experiments.
- Application of the generated power for wireless temperature and acceleration sensors.
- Investigation of an array of electromagnetic energy harvesters.

I. Investigation of hybrid harvester under harmonic excitation

Proposed Hybrid Piezo-Electromagnetic Energy Harvester

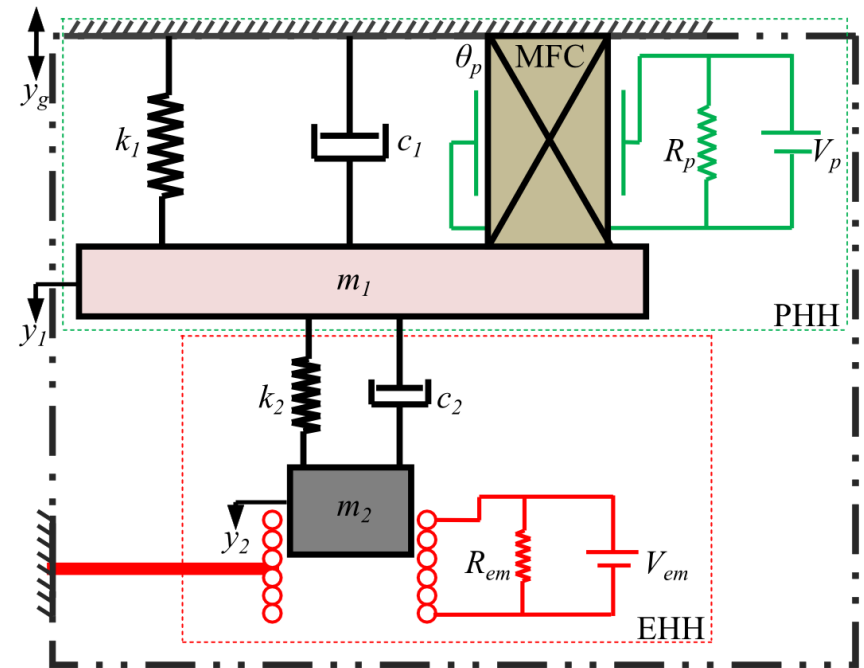
Harvesting energy using both piezoelectric and electromagnetic techniques in a single device.

Harvester Model



Physical Model

First mode
assumption



Equivalent Lumped Mass Model

Mathematical Model-Harmonic analysis

Piezo-electro-magneto-mechanical model of the hybrid harvester is derived using Newton's second law of motion, piezoelectric constitutive law and Faraday's law of induction,

$$m_1 \ddot{y}_{1r}(t) + c_1 \dot{y}_{1r}(t) - c_2 [\dot{y}_{2r}(t) - \dot{y}_{1r}(t)] + k_1 y_{1r}(t) - k_2 [y_{2r}(t) - y_{1r}(t)] + \theta_p V_p(t) = -\lambda m_1 \ddot{y}_g(t)$$

$$m_2 \ddot{y}_{2r}(t) + c_2 [\dot{y}_{2r}(t) - \dot{y}_{1r}(t)] + k_2 [y_{2r}(t) - y_{1r}(t)] = -m_2 \ddot{y}_g(t)$$

$$-\theta_p y_{1r}(t) + C_p \dot{V}_p(t) + \frac{V_p(t)}{\lambda R_p} = 0$$

$$V_{em}(t) + \theta_{em} \dot{y}_{2r}(t) = 0$$

Normalizing the equations w. r. t resonant frequency of the primary system,

$$[-\Omega^2 + 2\zeta_1 i\Omega + 2\mu\beta\zeta_2 i\Omega + \mu\beta^2 + 1]Y_{1r} - [2\mu\beta\zeta_2 i\Omega + \mu\beta^2]Y_{2r} + \left[\frac{\theta_p}{k_1}\right]V_p = \lambda\Omega^2 Y_g$$

$$-[2\beta\zeta_2 i\Omega + \beta^2]Y_{1r} + [-\Omega^2 + 2\beta\zeta_2 i\Omega + \beta^2]Y_{2r} = \Omega^2 Y_g$$

$$-\left[\frac{\theta_p}{C_p} \alpha i\Omega\right]Y_{1r} + [i\alpha\Omega + 1]V_p = 0$$

$$V_{em} = -\theta_{em} \omega_1 i\Omega Y_{2r}$$

$$\left\{ \begin{array}{l} \Omega = \frac{\omega}{\omega_1}; \alpha = \lambda\omega_1 C_p R_p \\ \beta = \frac{\omega_2}{\omega_1}; \mu = \frac{m_2}{m_1}; \kappa^2 = \frac{\theta_p^2}{k_1 C_p} \end{array} \right\}$$

Power output,

$$P_{PHH} = \frac{|V_p / \sqrt{2}|^2}{R_p};$$

$$P_{EHH} = \left| \frac{V_{em} / \sqrt{2}}{R_{em} + Rc} \right|^2 R_{em}$$

Total Power

$$P_{HH} = P_{PHH} + P_{EHH}$$

Experimental setup

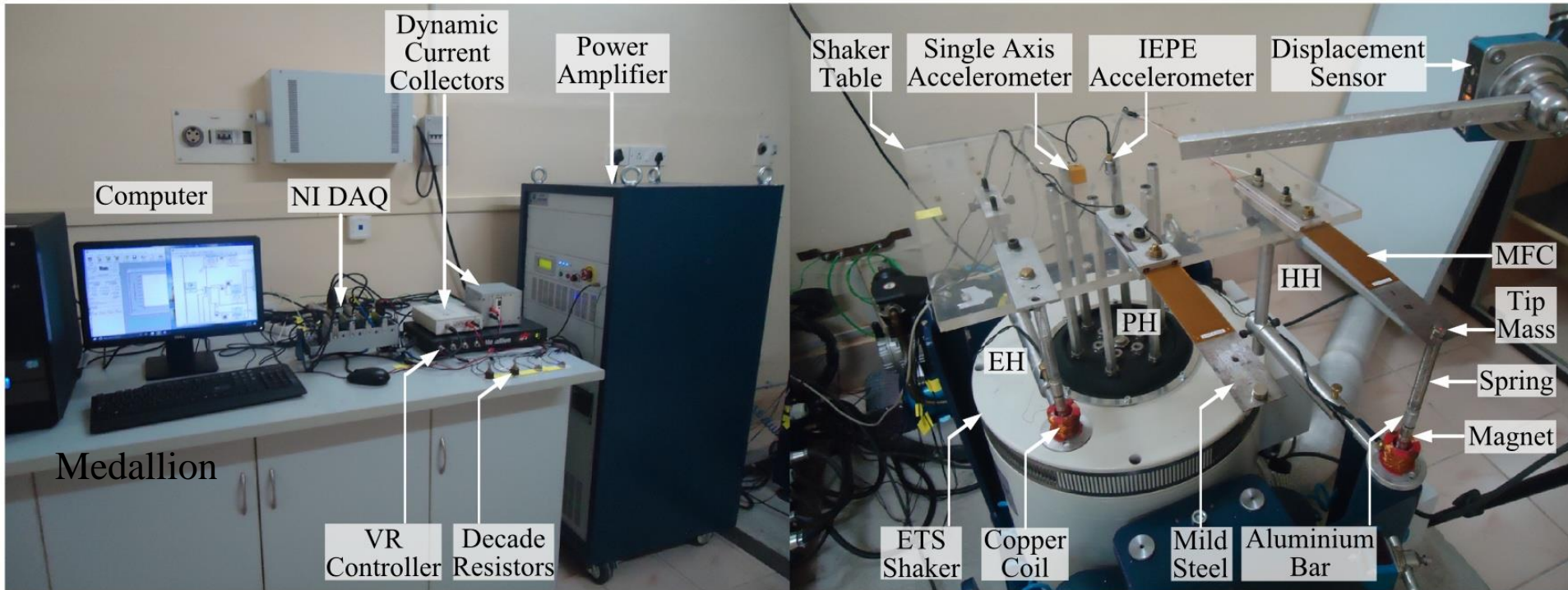


Fig: Photograph

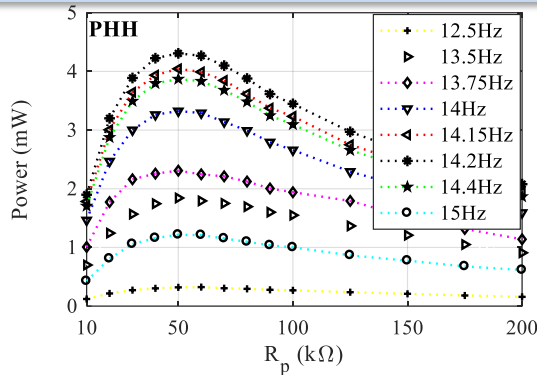
Table: Parameters and their values

Parameter	Value	Parameter	Value	Parameter	Value	Parameter	Value
L	160 mm	ρ	7800 kg/m ³	C_p	177.07 nF	B_m	1.1 T
b	32 mm	t_p	0.3 mm	R_p	50 k Ω	R_{em}	60 Ω
t_b	0.4 mm	C^E	15.857 Gpa	m_t	13.19 gm	ζ_1	0.0238
E	210 Gpa	d_{31}	170 pC/N	m_m	17.61 gm	ζ_2	0.0044

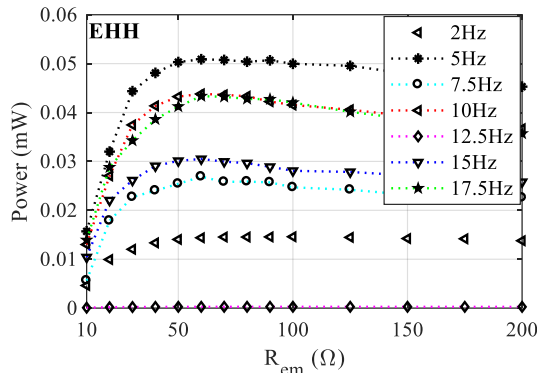
Harmonic analysis

- Frequency sweep
1-20Hz (Re 0.05Hz)
- Resistance sweep
PE:10-200k Ω , EM10-200 Ω
- Base excitation
1mm

Results: Harmonic analysis: contd...



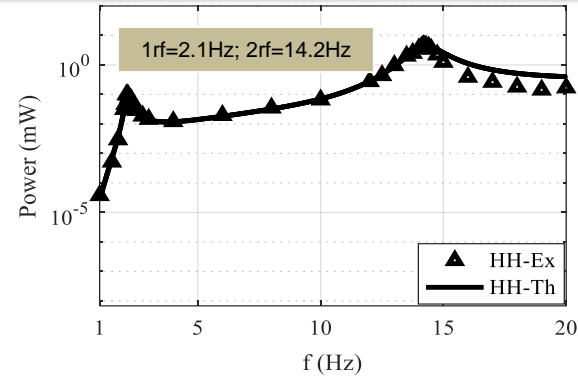
(a) Hybrid PE power with R_p



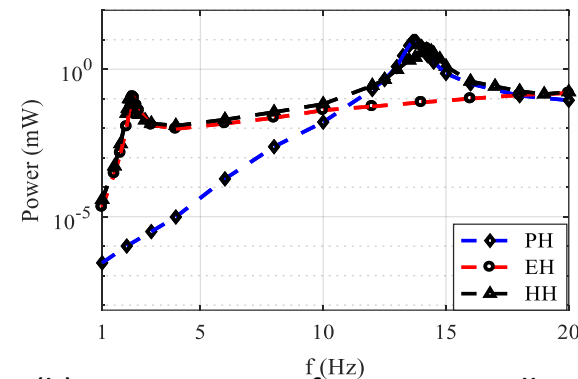
(b) Hybrid EM power with R_{em}

Figs : Variation of power with different set of load resistance

Optimal load resistance:
 $R_p = 50 \text{ k}\Omega$; $R_{em} = 60 \Omega$



(a) HH power-in test and in simulation

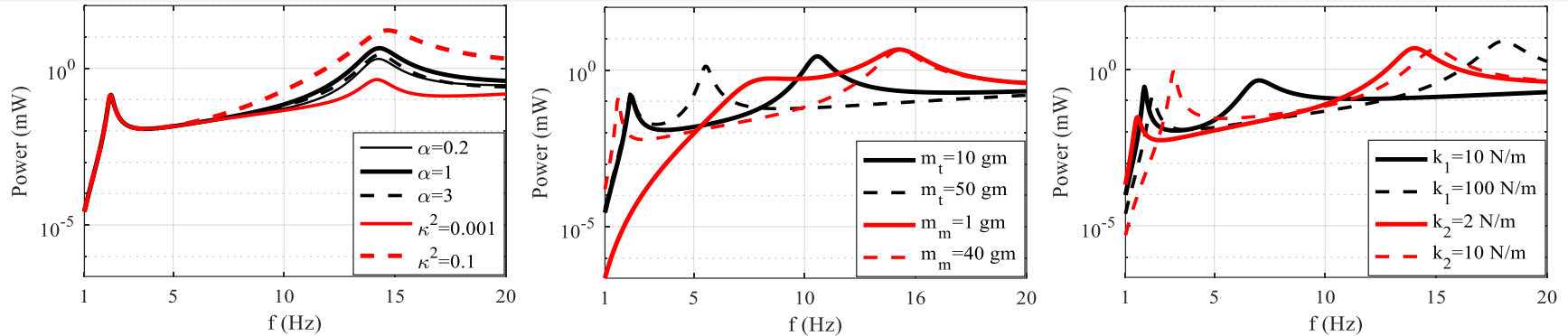


(b) Comparisons of experimentally obtained PH, EH and HH powers

Figs : Comparisons of Power outputs

The power harvested by the hybrid harvester exceeds the power obtained by standalone alternatives over a large region between the two resonating frequencies. This makes the hybrid harvester generate broadband power which is unachievable for standalone devices.

Results: Harmonic analysis: Parametric study



Figs: Power curves in hybrid energy harvester with different parameter values.

Table: Response of the hybrid harvester with respect to increase in parameters value

Parameter	First Resonating Frequency			Second Resonating Frequency			Total Band Width
	Band Width	Power	f_1 Shift	Band Width	Power	f_2 Shift	
α	No change	No change	No change	Increase	Max at $\alpha = 1$	Slightly right	Slightly increase
κ^2	Slightly decrease	Slightly decrease	No change	Increase	Increase	Right	Increase
m_t	Slightly increase	Slightly increase	No change	Decrease	Decrease	Left	Decrease
m_m	Decrease	Decrease	Left	No change	Decrease	Left	Increase
k_1	No significant	Slightly decrease	Slightly right	Increase	Increase	Right	Increase
k_2	Slightly decrease	Increase	Right	No change	No change	Right	Slightly decrease

Investigation of hybrid harvester under random excitation

Mathematical Model-Random analysis

EOM:

$$m_1 \ddot{y}_{1r}(t) + c_1 \dot{y}_{1r}(t) - c_2 [\dot{y}_{2r}(t) - \dot{y}_{1r}(t)] + k_1 y_{1r}(t) - k_2 [y_{2r}(t) - y_{1r}(t)] + \theta_p V_p(t) = -\lambda m_1 \ddot{y}_g(t)$$

$$m_2 \ddot{y}_{2r}(t) + c_2 [\dot{y}_{2r}(t) - \dot{y}_{1r}(t)] + k_2 [y_{2r}(t) - y_{1r}(t)] = -m_2 \ddot{y}_g(t)$$

$$-\theta_p y_{1r}(t) + C_p \dot{V}_p(t) + \frac{V_p(t)}{\lambda R_p} = 0$$

$$V_{em}(t) + \theta_{em} \dot{y}_{2r}(t) = 0$$

Gaussian

Power output:

$$P_{PHH}(t) = \frac{(V_p)^2}{R_p}; P_{EHH}(t) = \left[\frac{-\theta_{em} V_{em}}{[R_{em} + R_c]} \right]^2 R_{em}$$

Total Power

$$P_{HH}(t) = P_{PHH}(t) + P_{EHH}(t)$$

Based on the acceleration random input, the expected value arrived at

$$E[u_j(t)^2] = \frac{E[u_j(t)]^2}{c} = \int_{-\infty}^{\infty} \frac{1}{[\Delta(i\Omega)]^2} \frac{[H_j(\Omega)]^2}{\Omega^4} d\Omega$$

$$u_j(t) \longrightarrow y_{1r}(t), y_{2r}(t), V_p(t), V_{em}(t)$$

Solved by using classical Cramer's formula

Expected Power:

$$E[P_{HH}] = \frac{E[V_p^2]}{R_p} + \frac{E[V_{em}^2]}{[R_{em} + R_c]^2} R_{em}$$

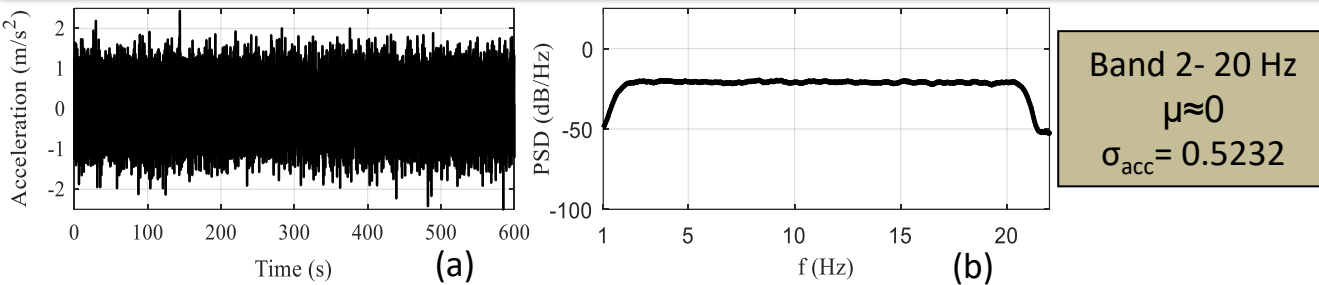
Random analysis

Base excitation:
Band limited
GWN

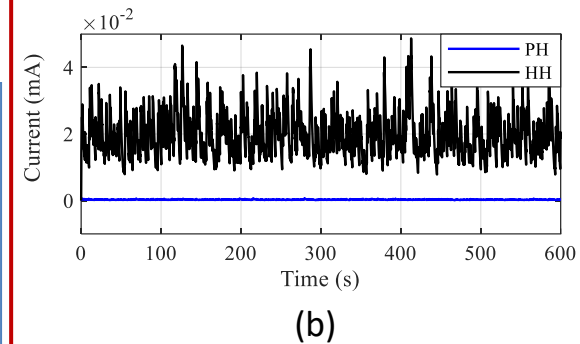
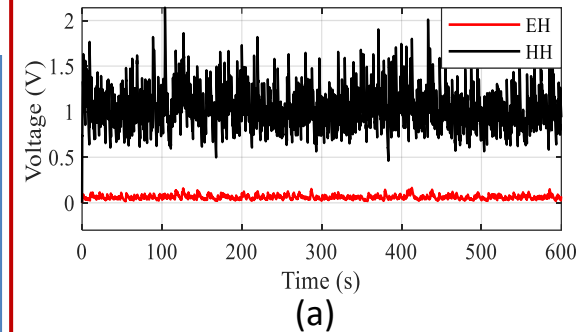
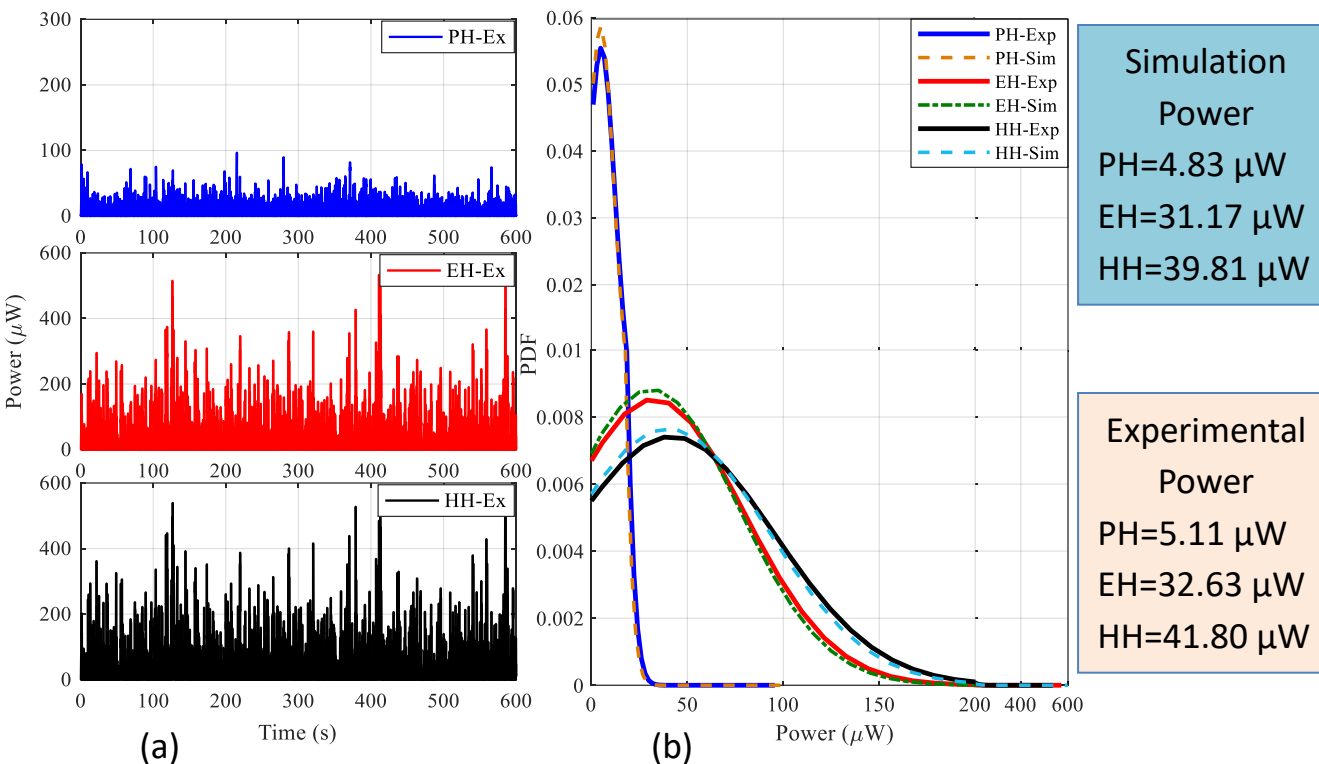
F_{band}: 2-5 Hz, 5-10 Hz, 10-15 Hz,
15-20 Hz, 2-10 Hz, 10-20 Hz,
2-20 Hz, 3.8-12.6Hz

Each frequency
band- σ_{acc}
is different

Results: Random analysis



Figs: Input excitation at the harvester base (a) recorded time histories (b) PSD



Figs : a) Comparisons of EH and HH voltages. (b) Comparisons of PH and HH currents.

Results: Random analysis: contd...

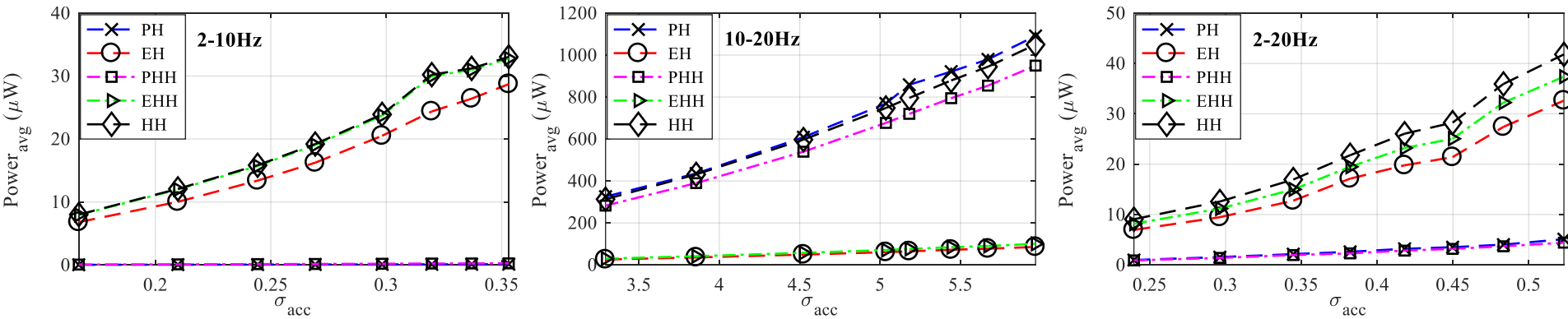


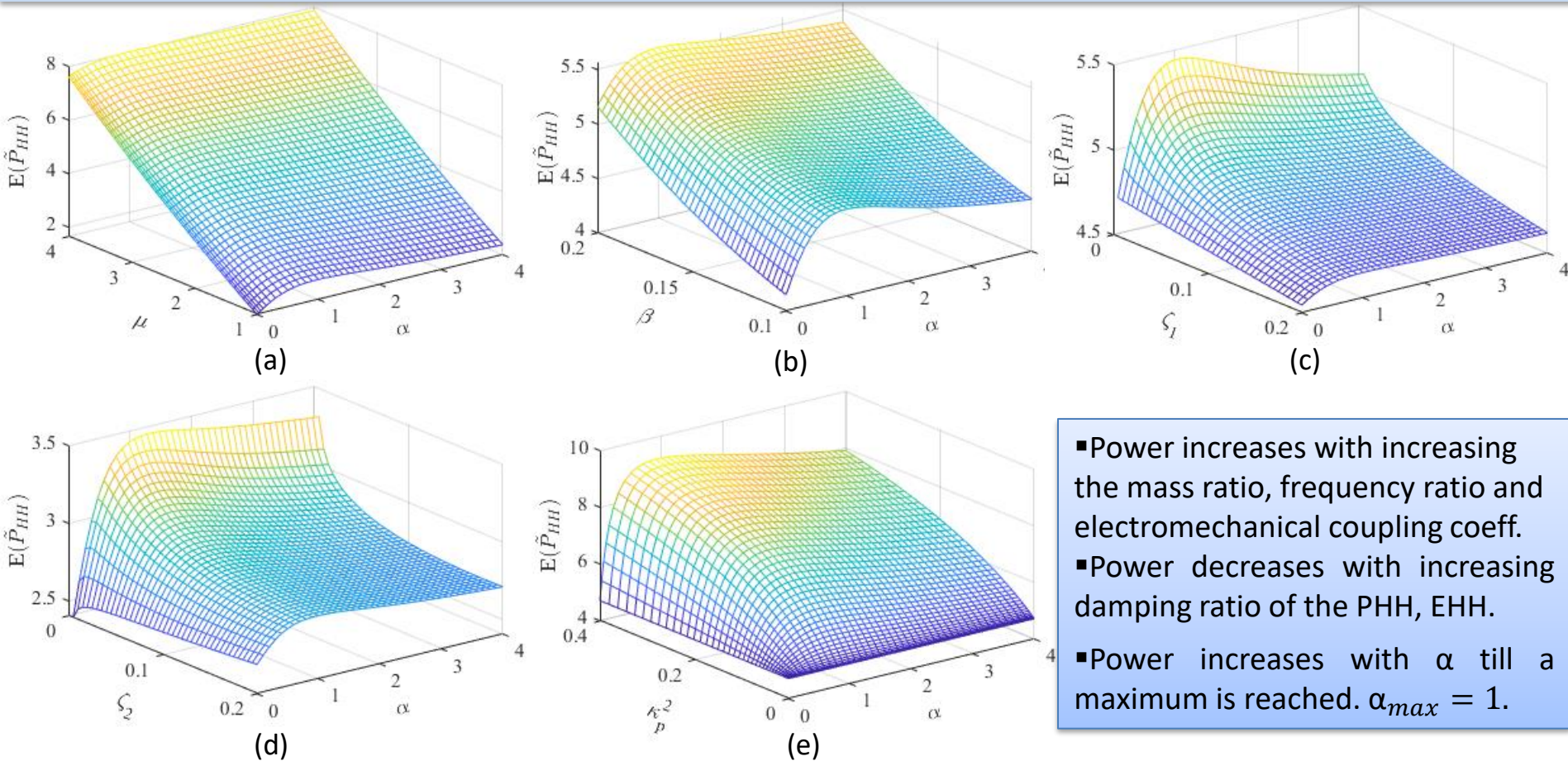
Fig: Average power versus standard deviation of input acceleration for different frequency bands

Table : Average power comparison of harvesters

Frequency Range	PH	EH	HH			PH+EH	HH
			PHH	EHH	Total		
2-5 Hz	I	III	II	IV	V	Low	High
5-10 Hz	I	III	II	IV	V	Low	High
10-15 Hz	V	I	III	II	IV	High	Low
15-20 Hz	III	II	IV	I	V	Low	High
2-10 Hz	I	III	II	IV	V	Low	High
10-20 Hz	V	I	III	II	IV	High	Low
2-20 Hz	II	III	I	IV	V	Low	High
3.8-12.6 Hz	II	III	I	IV	V	Low	High

- In the frequency bands between 2–10Hz, the EM harvesters perform better compared to PE harvesters.
- In the frequency bands between 10–20Hz, the PE harvester perform better than EM harvesters.
- HH performs better between the first and second resonating frequencies (3.8–12.6Hz) and in the wide bandwidth (2–20Hz).

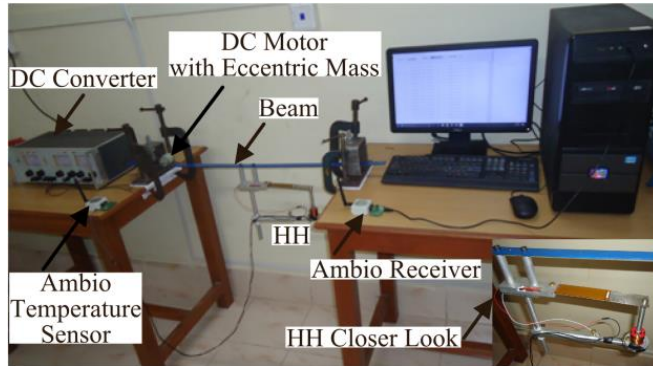
Results: Random analysis: Parametric study



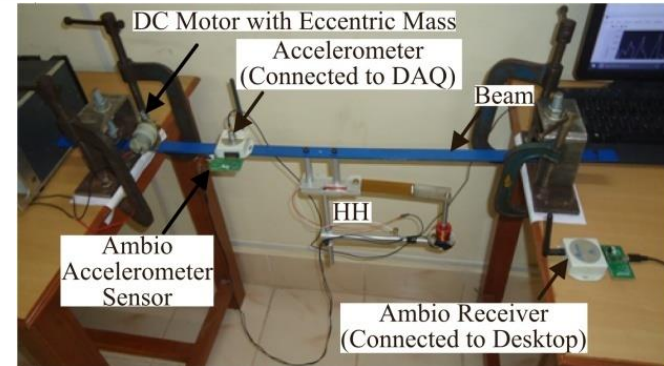
Figs: Variation of HH normalized mean power as a function of non-dimensional time constant, α and (a) mass ratio, μ (b) frequency ratio, β (c) damping ratio of the PHH, ζ_1 (d) damping ratio of the EHH, ζ_2 (e) non-dimensional electromechanical coupling coefficient. [Parameters other than the one being studied are considered as $\mu=2.6318$, $\beta=0.1643$, $\zeta_1=0.0238$, $\zeta_2=0.0613$, $\zeta_{em}=0.0569$, $\kappa_p^2=0.016$.]

Demonstration of hybrid power application

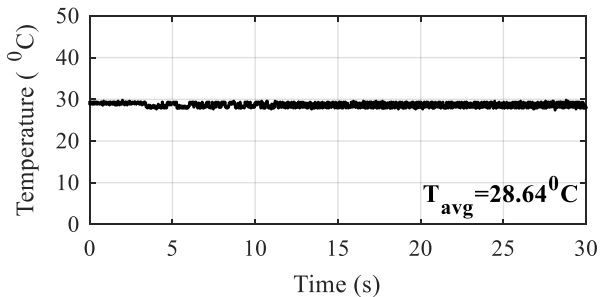
Demonstration :!



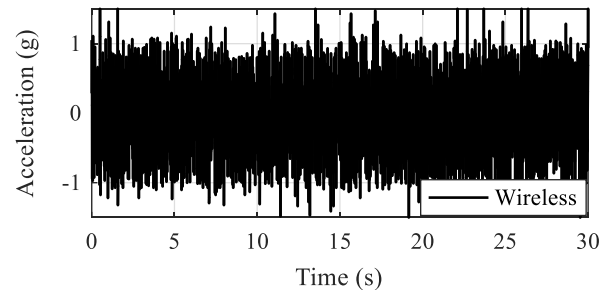
(a) Wireless temperature sensor powered from HH.



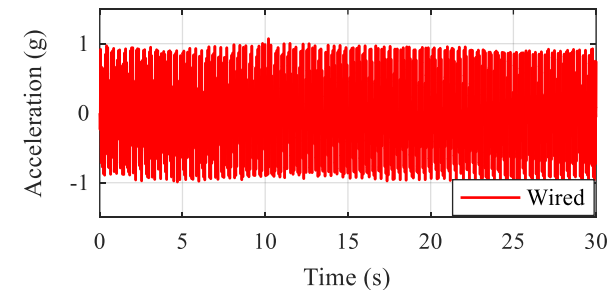
(b) Wireless accelerometer sensor powered from HH.



(c) Recorded temperature



(d) Recorded acceleration signal from wireless accelerometer.

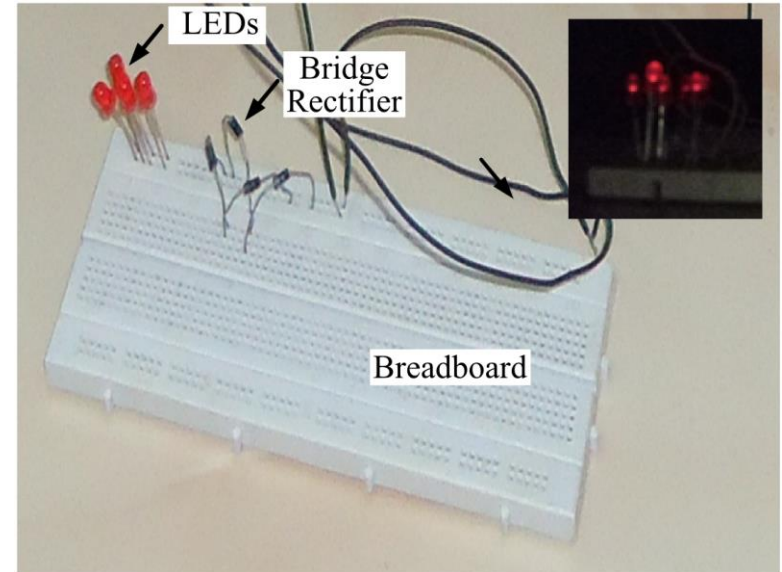
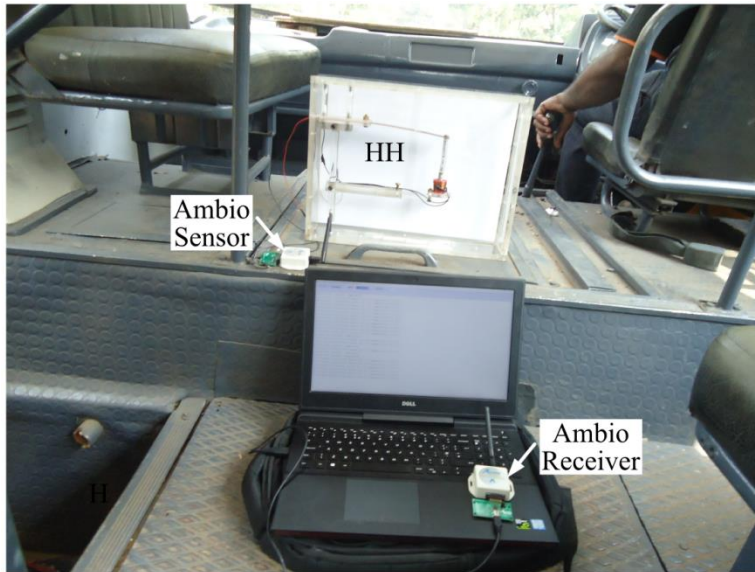


(e) Recorded acceleration signal from wired accelerometer.

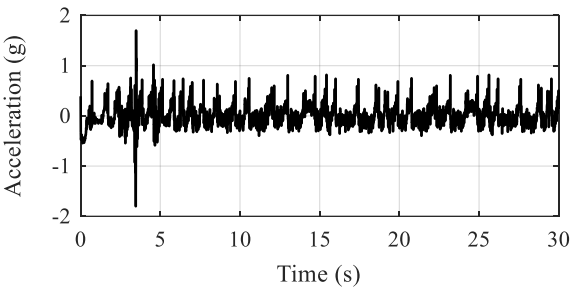
Figs. Experimental demonstration of wireless sensors on a beam powered through hybrid harvester (a) wireless temperature sensor, (b) wireless accelerometer sensor; recorded (c) temperature, (d) and (e) accelerations.

The wireless accelerometer is powered using HH for realizing the self-energized structural health monitoring

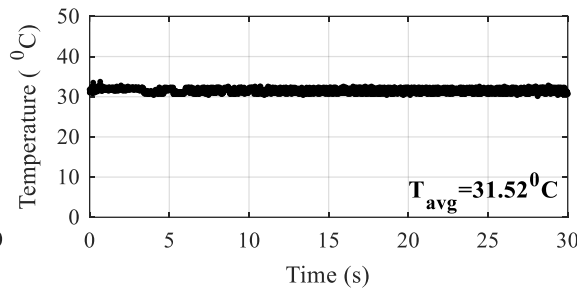
Demonstration: II&III



(a)



(b)



(c)

Figs: Experimental demonstration of wireless sensors on a bus powered through hybrid harvester (a) setup of wireless sensor, (b) recorded acceleration, (c) recorded temperature.

Fig: Experimental demonstration of light-emitting diodes powered through hybrid harvester.

The amount of power extracted from HH is adequate enough to power the wireless sensors and LEDs.

Demonstration: Video

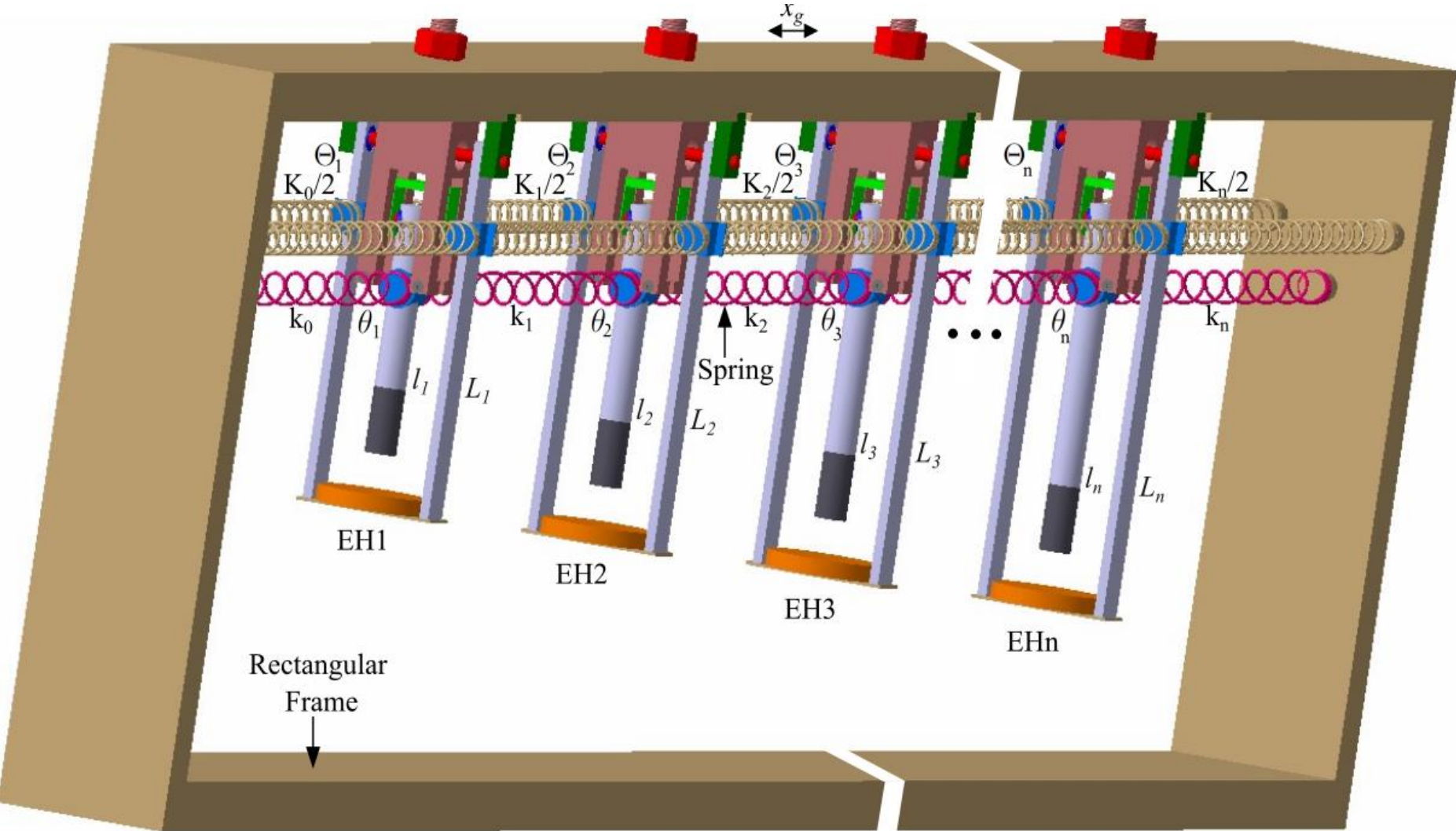


Summary of the first part

- This study reports experimental and theoretical studies of a coupled piezo-electromagnetic hybrid energy harvester.
- Simplified analytical model provides a good approximation to the experimental results.
- Hybrid harvester operates over a broad range of frequencies, compared to the narrow operating frequency range in standalone devices.
- Simultaneous production of high current and the high voltage is obtained due to the coupling of piezoelectric and electromagnetic transduction techniques.
- Parameters are discussed with the perspective of optimizing the magnitude and bandwidth of the harvested power.

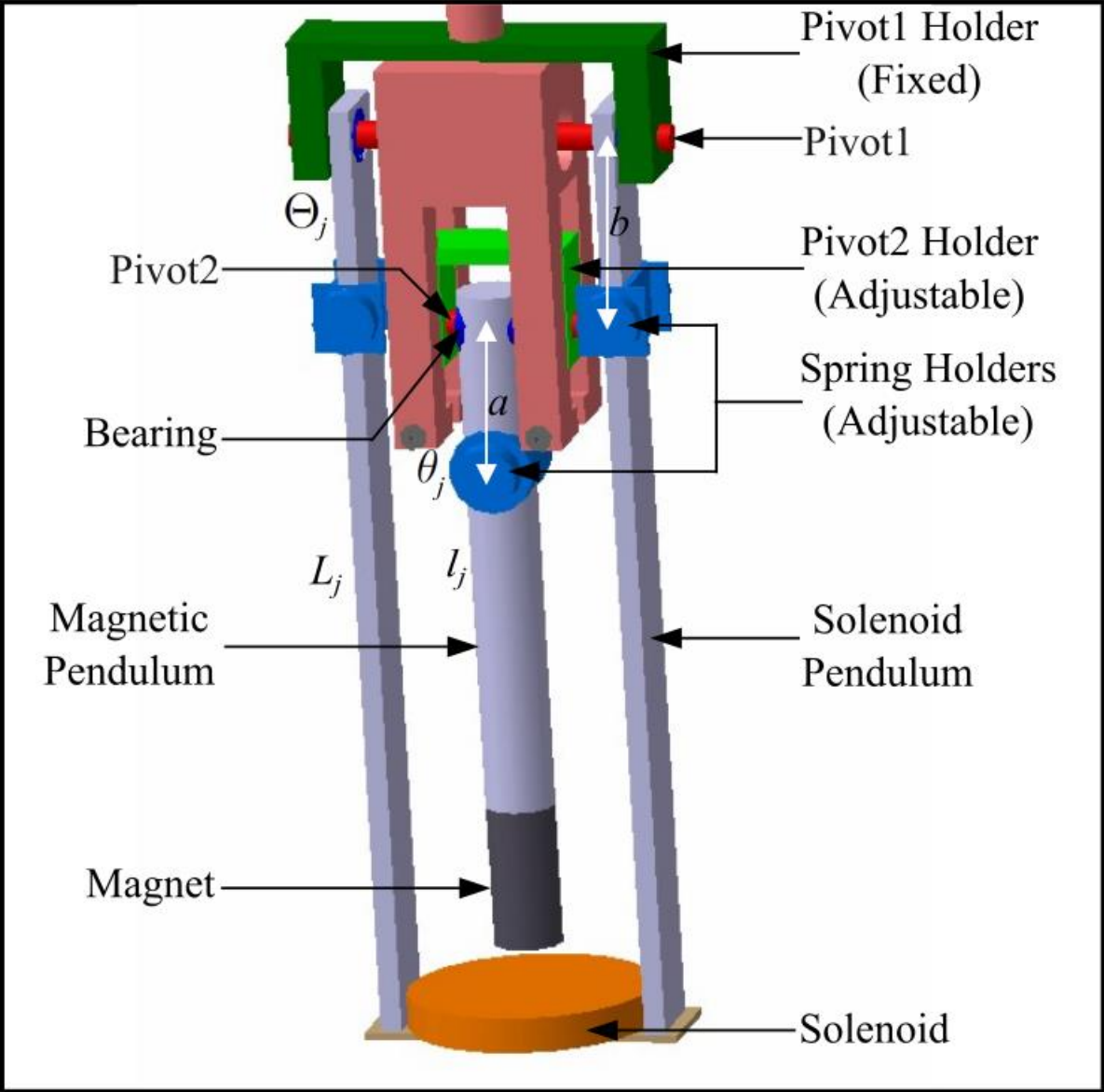
II. Investigation of an array of electromagnetic energy harvester

Electromagnetic energy harvester array



Physical Model

Harvester Model : contd...



Closeup view

Mathematical Model

Equations of motion of the n - pendulum array

$$m_j l_j^2 \ddot{\theta}_j(t) + c_j l_j^2 \dot{\theta}_j(t) + \bar{m}_j g l_j \sin \theta_j(t) + k_{j-1} a^2 [\sin \theta_j(t) - \sin \theta_{j-1}(t)] \cos \theta_j(t) \\ + k_j a^2 [\sin \theta_j(t) - \sin \theta_{j+1}(t)] \cos \theta_j(t) = -m_j l_j \ddot{x}_g(t) \cos \theta_j(t); \\ \{ j=1, 2, 3 \dots n, \quad \theta_{j-1}(t) = 0; \text{ at } j = 1, \quad \theta_{j+1}(t) = 0; \text{ at } j = n$$

$$M_j L_j^2 \ddot{\Theta}_j(t) + C_j L_j^2 \dot{\Theta}_j(t) + \bar{M}_j g L_j \sin \Theta_j(t) + K_{j-1} b^2 [\sin \Theta_j(t) - \sin \Theta_{j-1}(t)] \cos \Theta_j(t) \\ + K_j b^2 [\sin \Theta_j(t) - \sin \Theta_{j+1}(t)] \cos \Theta_j(t) = -M_j L_j \ddot{x}_g(t) \cos \Theta_j(t); \\ \{ j=1, 2, 3 \dots n, \quad \Theta_{j-1}(t) = 0; \text{ at } j = 1, \quad \Theta_{j+1}(t) = 0; \text{ at } j = n$$

Nondimensional form

$$\mu_j \alpha_j^2 \ddot{\theta}_j' + 2\zeta_j \mu_j \alpha_j^2 \Omega_j \dot{\theta}_j' + \bar{\mu}_j \alpha_j \sin \theta_j' + \beta_{j-1} [\sin \theta_j' - \sin \theta_{j-1}'] \cos \theta_j' \\ + \beta_j [\sin \theta_j' - \sin \theta_{j+1}'] \cos \theta_j' = \mu_j \alpha_j \Omega^2 \lambda \sin(\Omega \tau) \cos \theta_j'; \\ \{ \theta_{j-1}' = 0; \text{ at } j = 1, \quad \theta_{j+1}' = 0; \text{ at } j = n$$

$$\hat{\mu}_j \hat{\alpha}_j^2 \ddot{\Theta}_j' + 2\hat{\zeta}_j \hat{\mu}_j \hat{\alpha}_j^2 \hat{\Omega}_j \dot{\Theta}_j' + \hat{\mu}_j \hat{\alpha}_j \sin \Theta_j' + \hat{\beta}_{j-1} (\sin \Theta_j' - \sin \Theta_{j-1}') \cos \Theta_j' \\ + \hat{\beta}_j (\sin \Theta_j' - \sin \Theta_{j+1}') \cos \Theta_j' = \hat{\mu}_j \hat{\alpha}_j \Omega^2 \lambda \sin(\Omega \tau) \cos \Theta_j'; \\ \{ \Theta_{j-1}' = 0; \text{ at } j = 1, \quad \Theta_{j+1}' = 0; \text{ at } j = n$$

Mathematical Model : contd...

Nondimensional parameters :

$$\mu_j = \frac{m_j}{m_1}; \hat{\mu}_j = \frac{M_j}{m_1}; \bar{\mu}_j = \frac{\bar{m}_j}{\bar{m}_1}; \hat{\bar{\mu}}_j = \frac{\bar{M}_j}{\bar{m}_1}; \alpha_j = \frac{l_j}{l_1}; \hat{\alpha}_j = \frac{L_j}{l_1};$$

$$\zeta_j = \frac{c_j}{2m_1\omega_1}; \hat{\zeta}_j = \frac{C_j}{2m_1\omega_1}; \beta_j = \frac{k_j a^2}{m_1 l_1^2 \omega_1^2}; \hat{\beta}_j = \frac{K_j b^2}{m_1 l_1^2 \omega_1^2}; \lambda = \frac{X_g}{l_1}; \Omega = \frac{\omega}{\omega_1}$$

Table: List of harvester configurations with different coupling combinations

SaMH	UC		UC SaMH
	C&UG		C&UG SaMH
	C&G		CG SaMH
SaSH	UC		UC SaSH
	C&UG		C&UG SaSH
	C&G		C&G SaSH
MSH	UC SH	UC MH	UC MSH
		C&UG MH	UC SH with C&UG MH
		CG MH	UC SH with C&G MH
	C&UG SH	UC MH	C&UG SH with UC MH
		C&UG MH	C&UG MSH
		CG MH	C&UG SH with C&G MH
	C&G SH	UC MH	C&G SH with UC MH
		C&UG MH	C&G SH with C&UG MH
		CG MH	C&G MSH

Total Power

$$|P_t| = \frac{1}{2} \sum_{j=1}^4 \Omega^2 \zeta_{emj} (\hat{\alpha}_j \Theta_j - \alpha_j \theta_j)^2$$

Coupled and ungrounded (C&UG)

$$\beta_0, \beta_n, \hat{\beta}_0, \hat{\beta}_n = 0$$

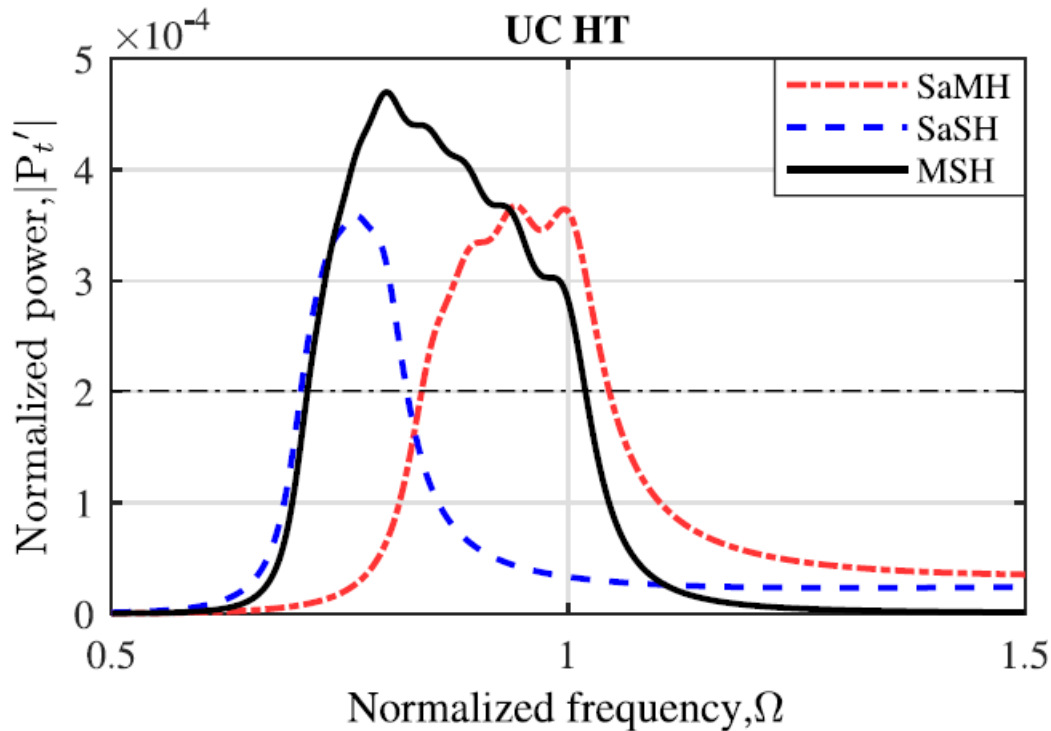
Uncoupled and ungrounded (UC)

$$\beta_q, \hat{\beta}_q = 0$$

Results:

Linear dynamics

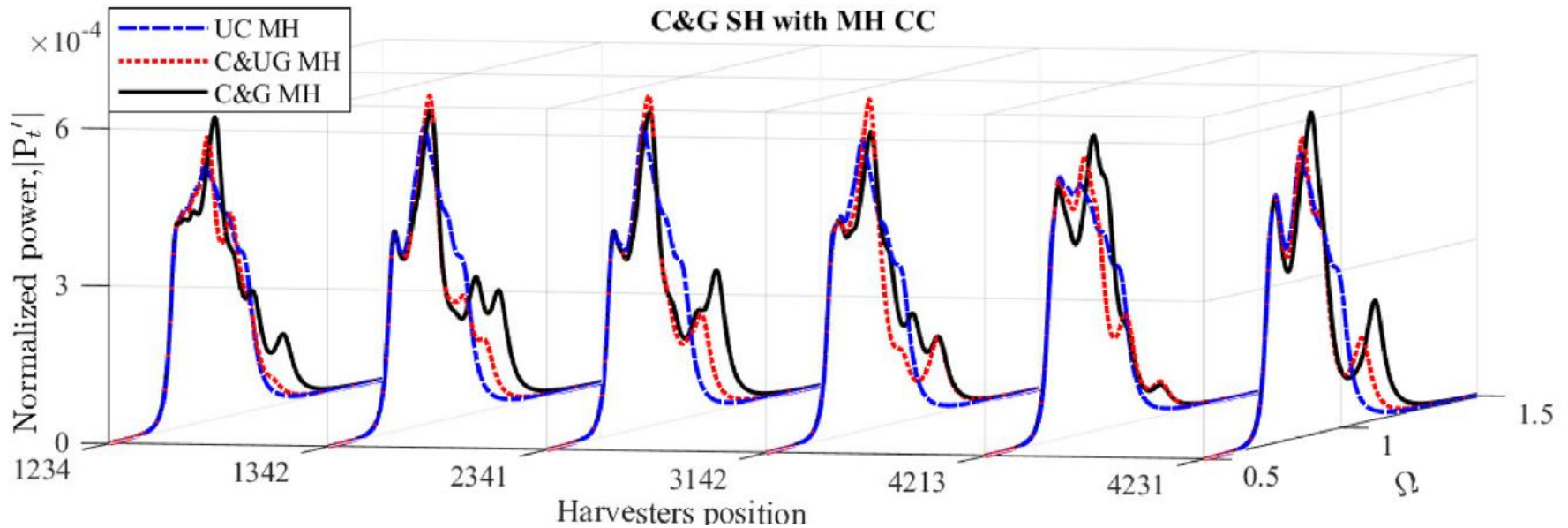
$$\lambda = 0.01$$



Power @ 2×10^{-4} , the bandwidth of the UC MSH increases by 45.57% compared to UC SaMH and by 162.07% compared to UC SaSH.

Figs : Frequency response of the uncoupled harvesters' normalized power [at $\lambda = 0.01$].

Results: contd...



Figs : Comparisons of the total harvested normalized power of C&G SH with MH CC across various positions of the harvesters. [Note: only six of them are shown for visual clarity]

Each harvester in the harvesting array is designed to have a different resonant frequency, so their performance varies depending on their location when they are coupled.

Spring with equal stiffness → $n!/2$ possible combinations

1234	1423	1342	2314	2143	2341	3214	3142	3421	4213	4312	4231
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Results: contd...

Table : Comparisons of relative power density

Coupling ↓ Combinations Harvesters Position →		Relative Power Density (%)												
		1234	1423	1342	2314	2143	2341	3214	3142	3421	4213	4312	4231	
MH	UC	100												
	C&UG	100.31	101.33	100.87	101.56	101.16	101.08	101.19	101.77	100.52	100.99	100.74	100.92	
	C&G	109.84	111.82	112.75	108.87	109.36	112.97	107.10	110.00	110.96	106.89	108.02	110.47	
SH	UC	66.35												
	C&UG	66.40	66.56	66.48	66.58	66.52	66.51	66.53	66.62	66.43	66.50	66.46	66.49	
	C&G	69.25	69.64	69.88	69.01	69.18	69.91	68.65	69.28	69.51	68.62	68.88	69.34	
MSH	UC SH	UC MH	126.59											
		C&UG MH	126.37	126.34	126.25	126.57	126.47	126.17	126.30	126.62	126.27	126.38	126.52	126.51
		C&G MH	139.22	140.48	141.84	137.30	137.83	141.85	135.54	138.19	140.11	135.50	136.90	139.58
	C&UG SH	UC MH	127.12	129.98	128.80	129.88	128.75	129.05	128.78	130.47	127.99	129.18	128.44	128.97
		C&UG MH	126.79	127.55	127.19	127.90	127.52	127.31	127.52	128.00	126.92	127.33	127.22	127.32
		C&G MH	139.63	141.88	143.15	138.09	138.76	143.38	136.03	139.42	140.93	135.87	137.30	140.32
	C&G SH	UC MH	127.63	130.28	130.57	129.90	129.07	129.85	128.40	130.90	129.13	128.86	128.28	129.77
		C&UG MH	127.02	127.35	127.05	127.99	127.61	127.05	127.61	128.00	127.04	127.47	127.39	127.40
		C&G MH	140.86	142.96	144.44	139.05	139.80	144.59	136.82	140.41	142.21	136.65	138.27	141.47

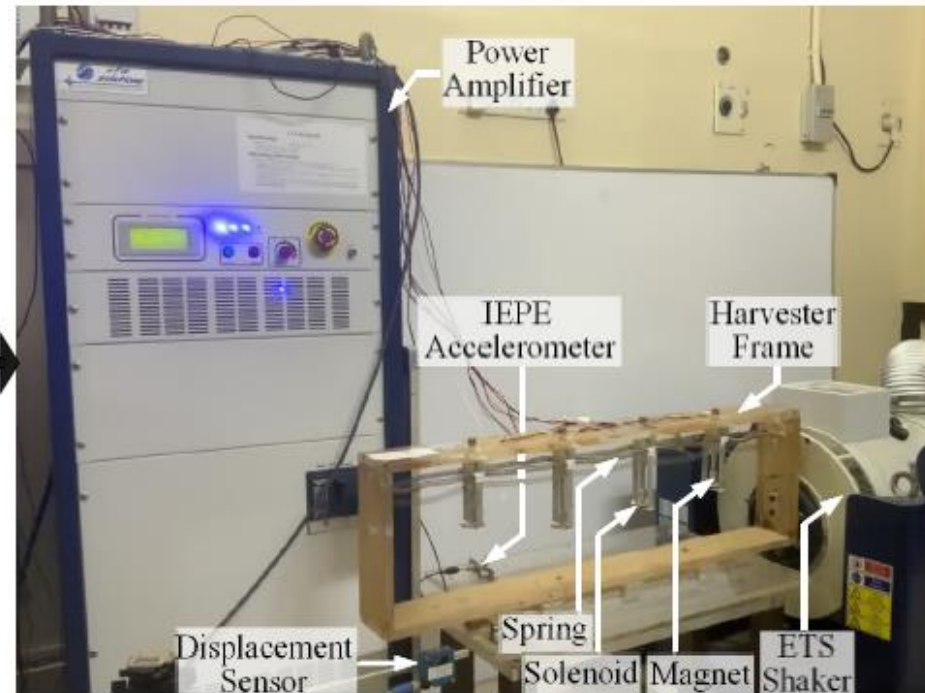
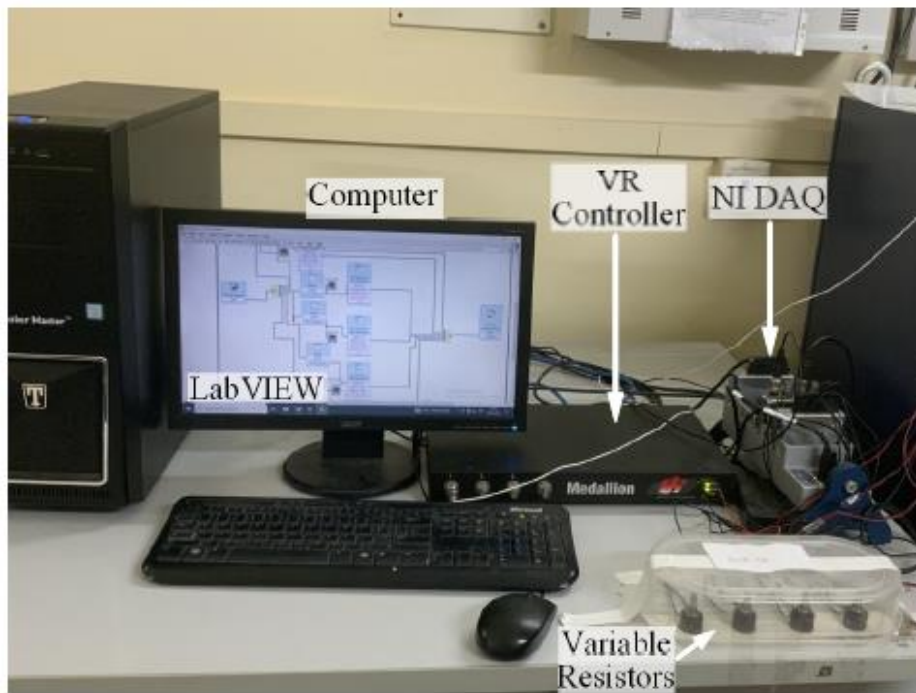
$$RPD_H = \left(\frac{PD_H}{PD_{UCSaMH}} \right) 100 \%$$

PD_H - power density of the respective harvester array.

PD_{UCSaMH} - power density of the uncoupled MH array.

C&G SaMH, C&G SaSH, and C&G MH with SH CC in MSH give the maximum power density at the harvester position of 2341, and all other uncoupled/coupled combinations give the maximum power density at the harvester position of 3142.

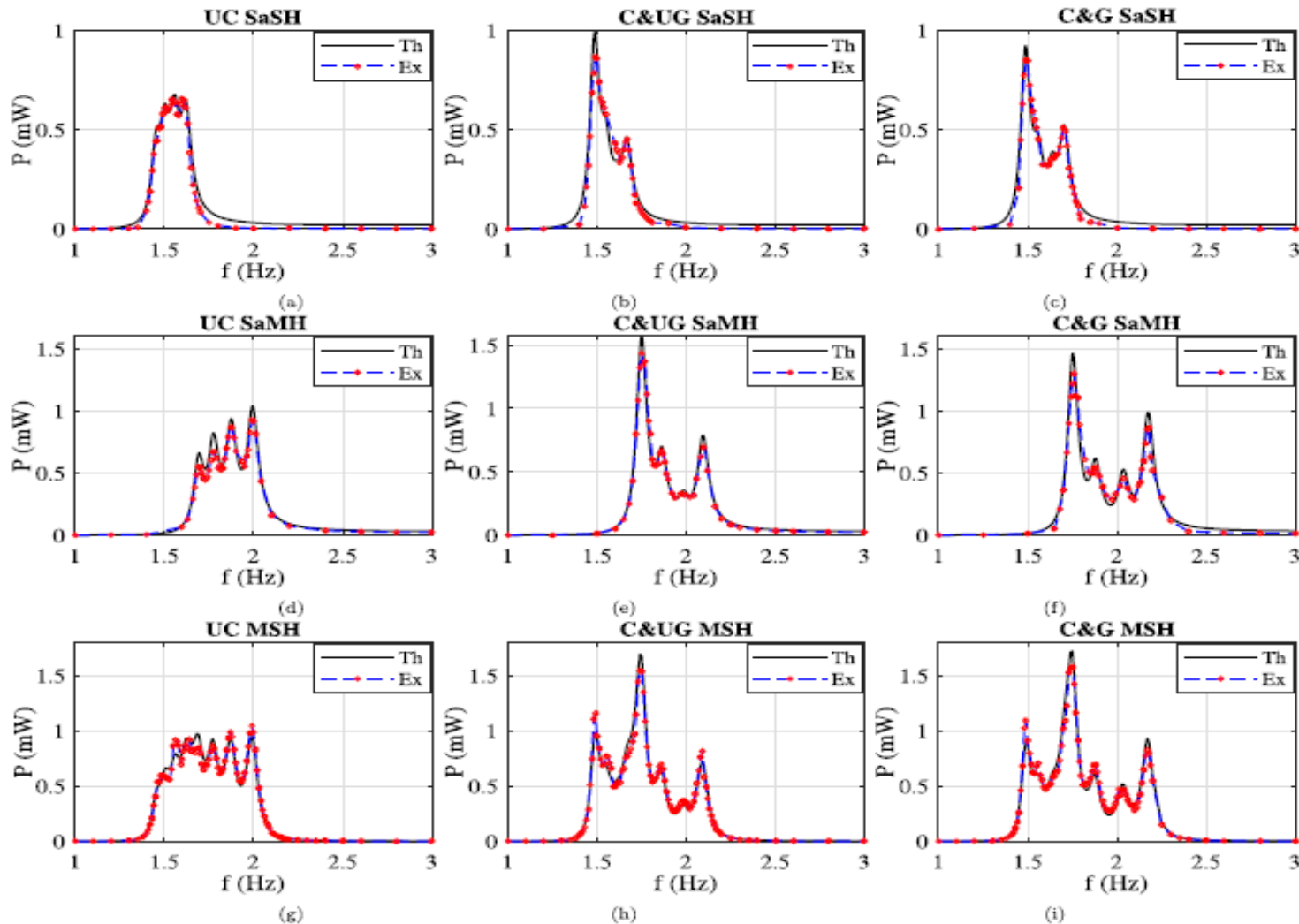
Experimental setup



Parameters and their values

Description	Value			
Length of the MP, l_j (mm)	70	80	90	100
Mass of the magnet part, m_{m_j} (gm)	17.6	17.6	17.6	17.6
Mass of the MP rod, m_{r_j} (gm)	16.6	18.6	20.6	22.6
Mechanical damping of the MP, c_{me_j} (Ns/m)	0.0064	0.0059	0.0056	0.0052
Length of the SP, L_j (mm)	110	120	130	140
Mass of the solenoid bobbin, M_{s_j} (gm)	17.2	17.2	17.2	17.2
Mass of the SP strips, M_{st_j} (gm)	24.6	26.6	28.6	30.6
Mechanical damping of the SP, C_{me_j} (Ns/m)	0.0071	0.0069	0.0066	0.0064

Multiple electromagnetic hybrid harvester



- Frequency sweep 1-3Hz
- Base excitation 0.7mm

Figs: Comparison of simulated and experimentally obtained powers for different types of harvesters with various coupling combinations.

Conclusions

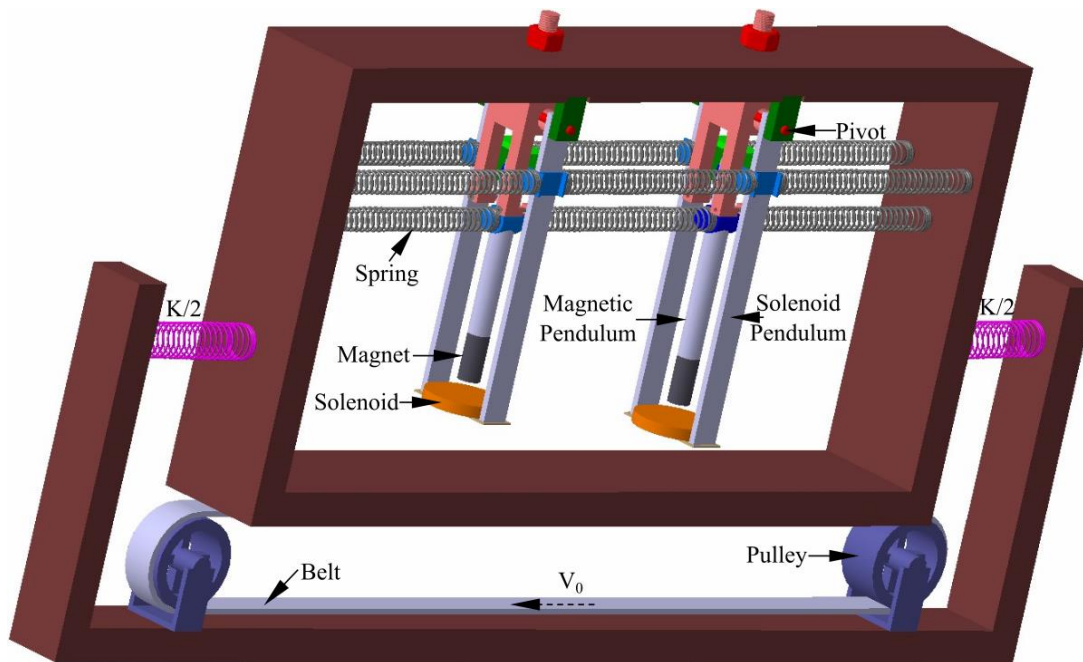
- This study reports a novel design of a pendulum-based array of electromagnetic vibration energy harvester.
- The investigations include three types of harvesters with different mechanical coupling configurations.
- The position of the harvesters was varied and analyzed.
- It is observed that, coupled and grounded magnetic-solenoid harvester gives the better performance.
- The experimental observations show good agreement with the theoretical results.

Extended Research

1. Investigation of an array of pendulum-based electromagnetic nonlinear vibration energy harvester

Base excitation
3.5mm

2. Analysis of dual pendulum-based friction induced electromagnetic vibration energy harvester array



Stick-slip behavior
Stribeck model

THANK YOU